



KUWAIT 3RD FLOW MEASUREMENT TECHNOLOGY CONFERENCE

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A Subsidiary of Kuwait Petroleum Corporation



Improving Flow Measurement in Heavy Oils with a Multiparameter System





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THE NEED

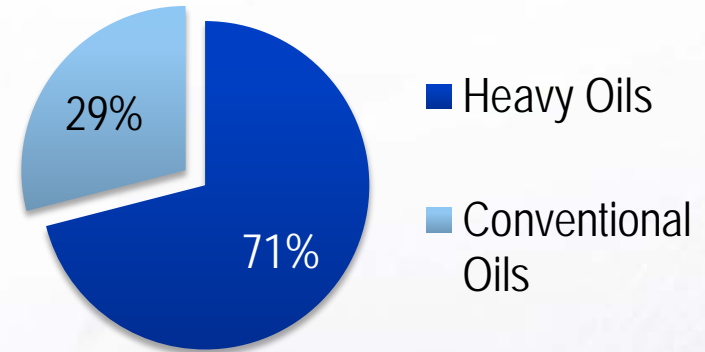
Increasing world energy demands

Measurement issues in low Reynolds numbers

Non-linear Performance Index in low Re

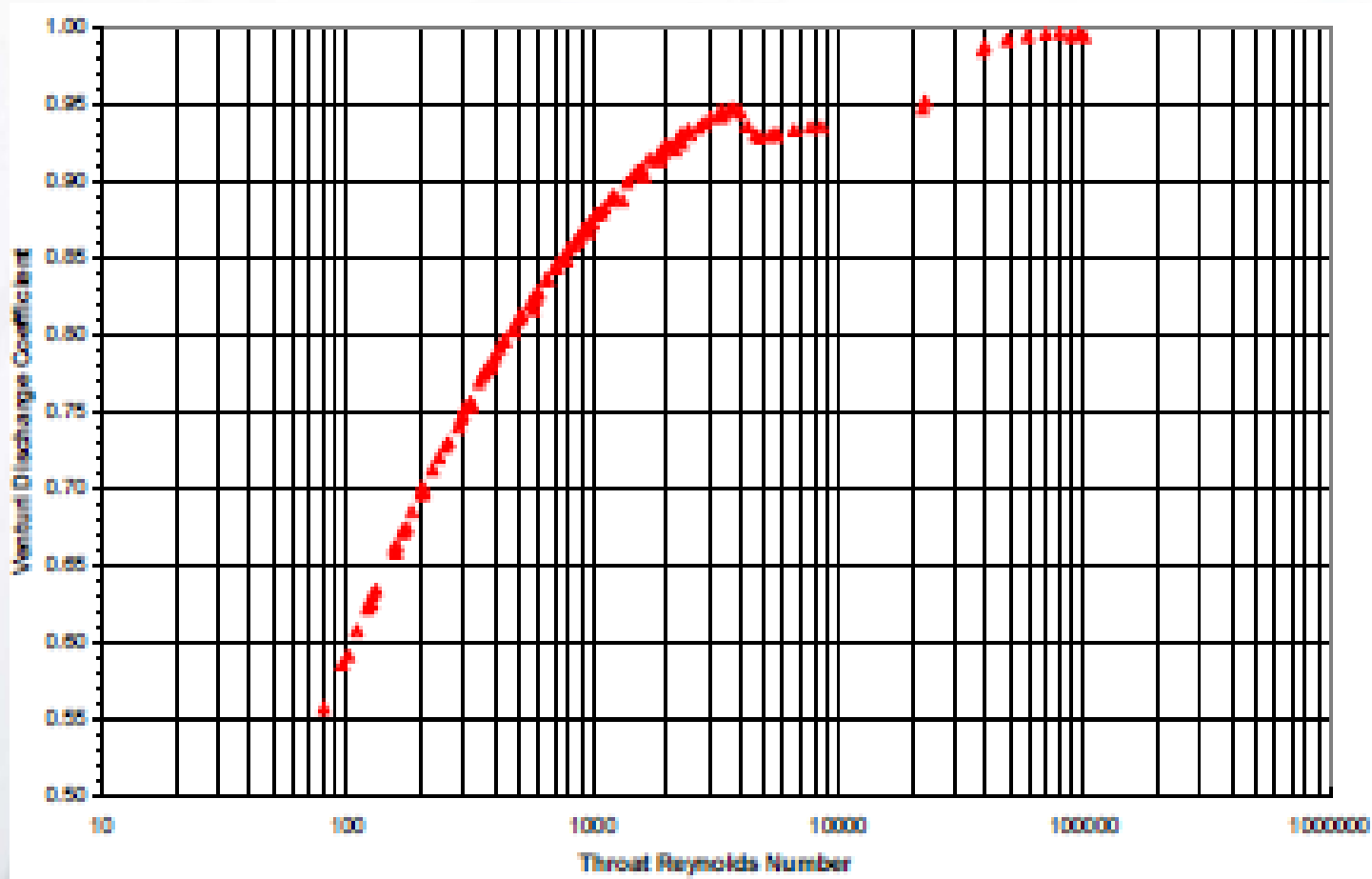
Need for improved performance

Remaining Reserves

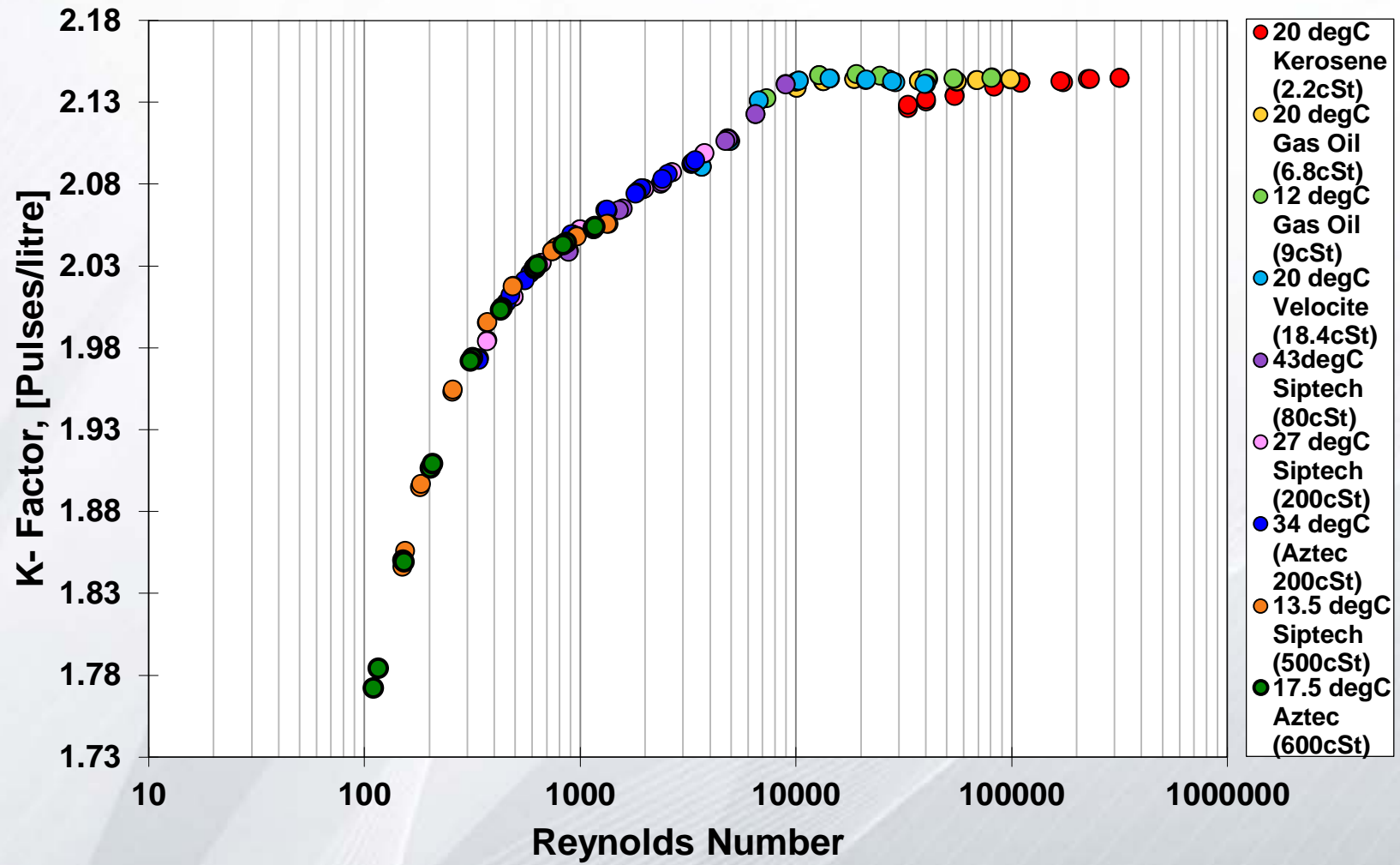


LIGHT OIL	45.5°
	31.1°
MEDIUM	30.2°
	22.3°
HEAVY	21.5°
	10.0°
EXTRA-HEAVY	6.5°
	0.1°

Venturi Performance

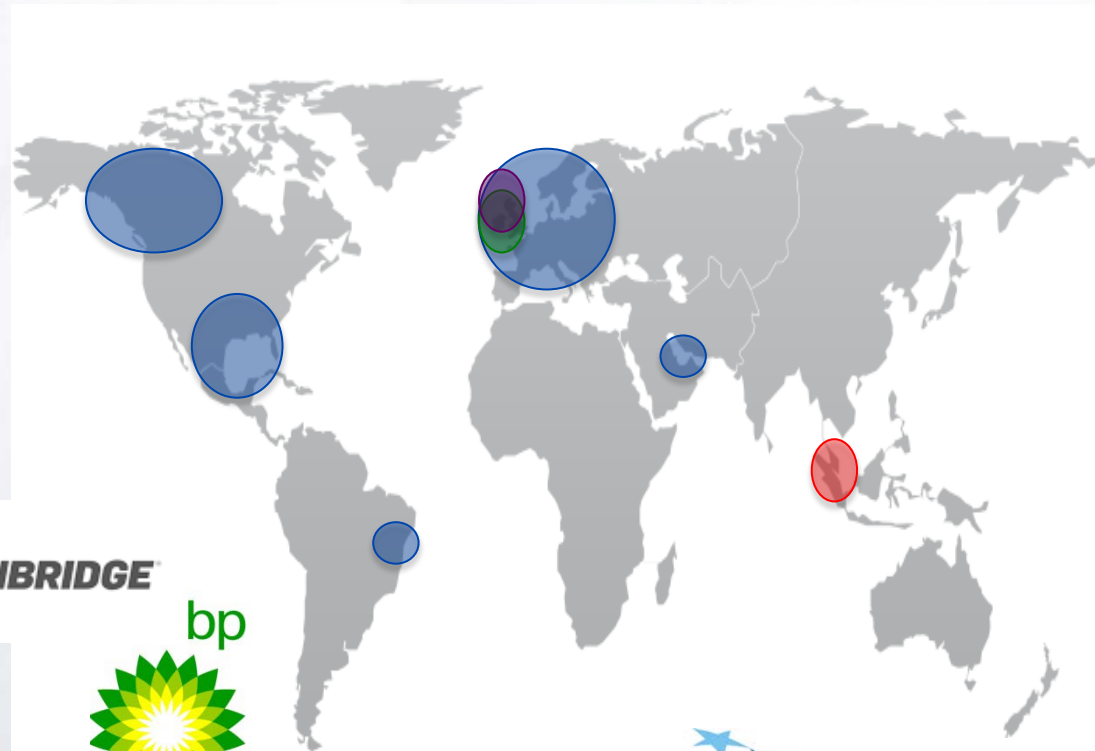


Turbine Performance



Market Study

8 countries visited, another 10 countries contacted, >100 people engaged
>30 end users, >50 manufacturers, >20 stakeholders



Bunker Fuel

Food & Drink

Chemical

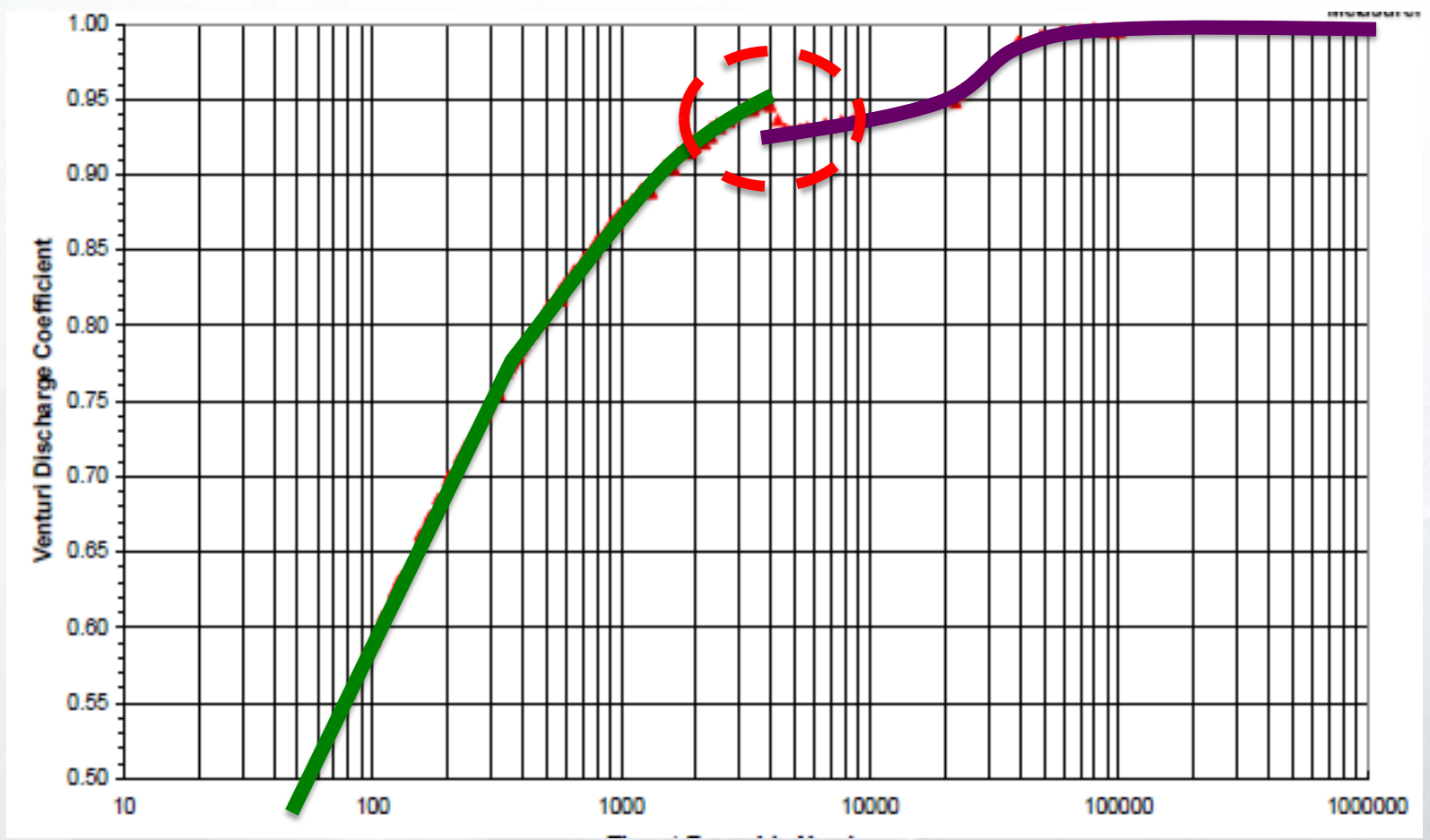


Market Study Outputs

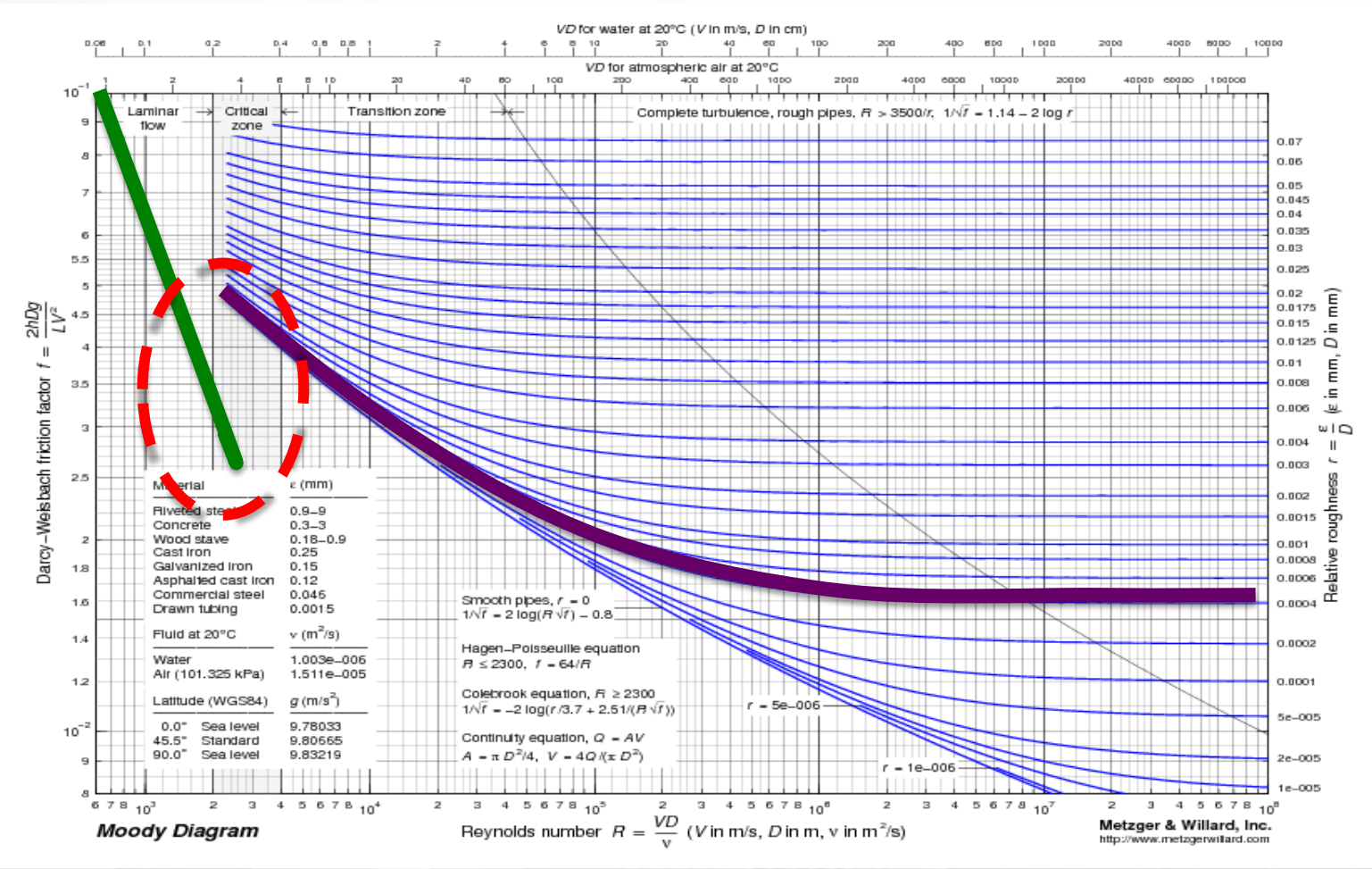
- Better understanding of Fluid Mechanics
- Cost is critical but not the accuracy
- Low cost and 2% accuracy is acceptable
- There a gap in market for the above
- DP meters has a good potential if developed for low Re applications



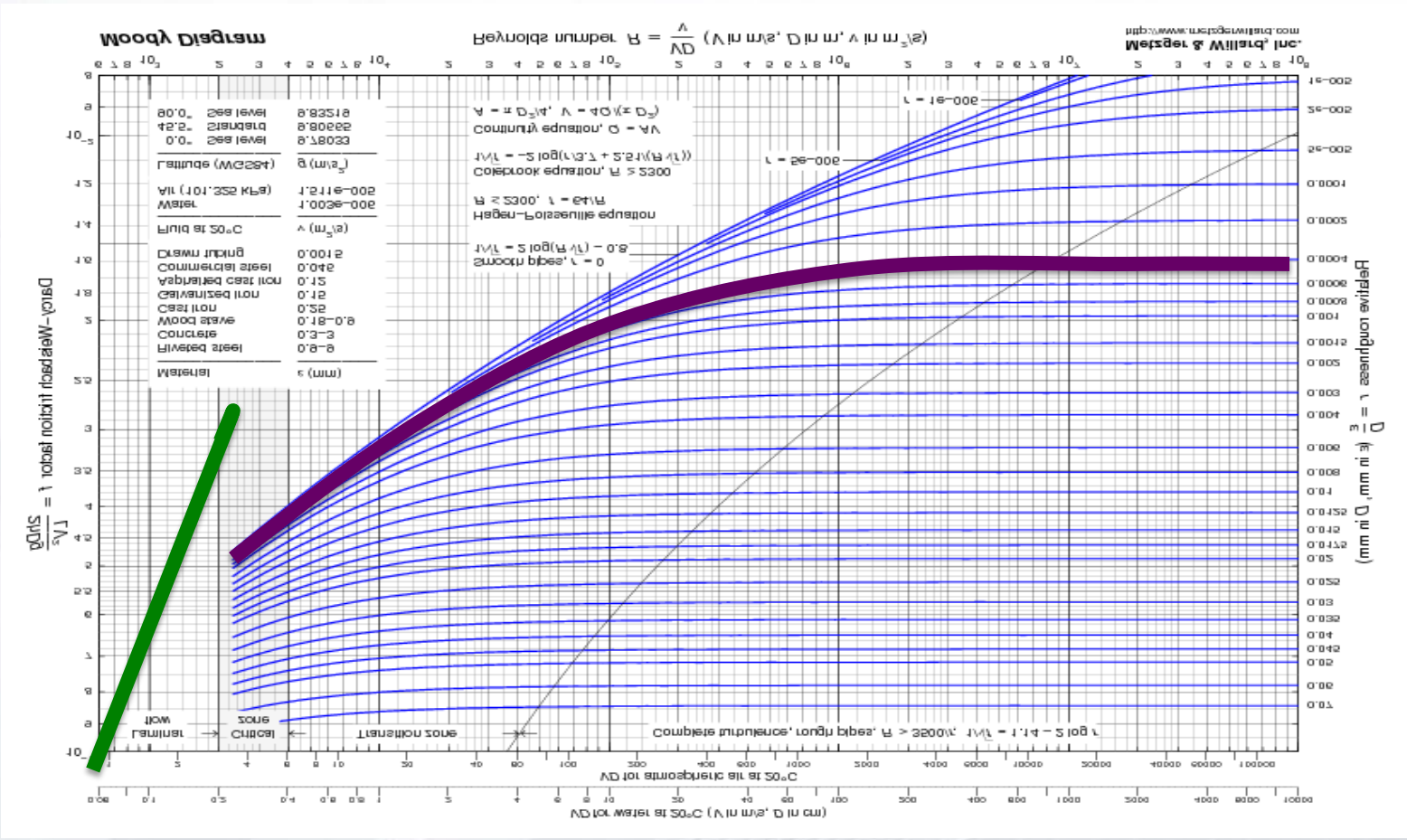
Performance



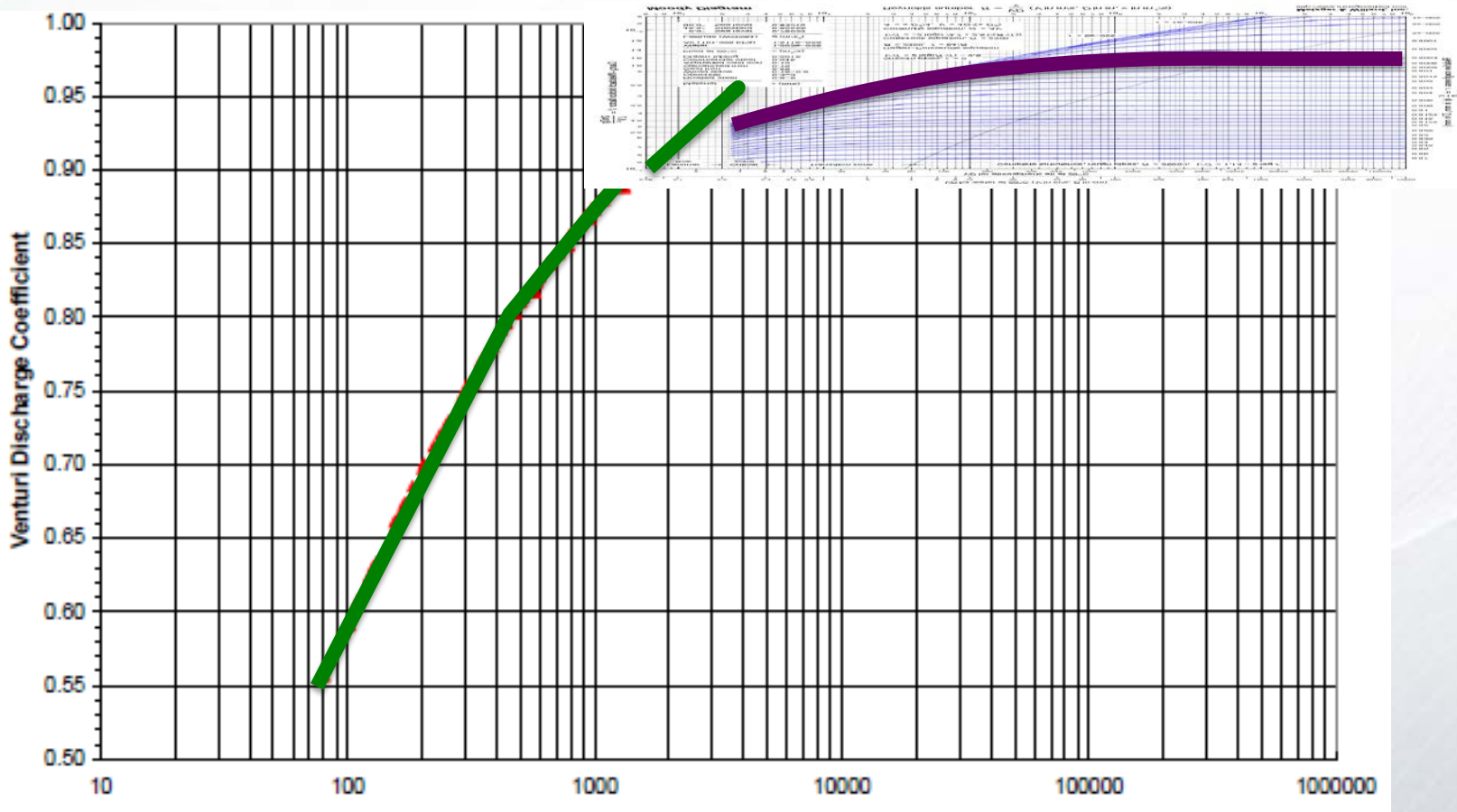
Moody Plot



Invert



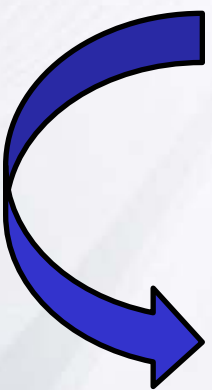
Overlap



Theory (1)

Consider the fundamental theories used in fluid mechanics:

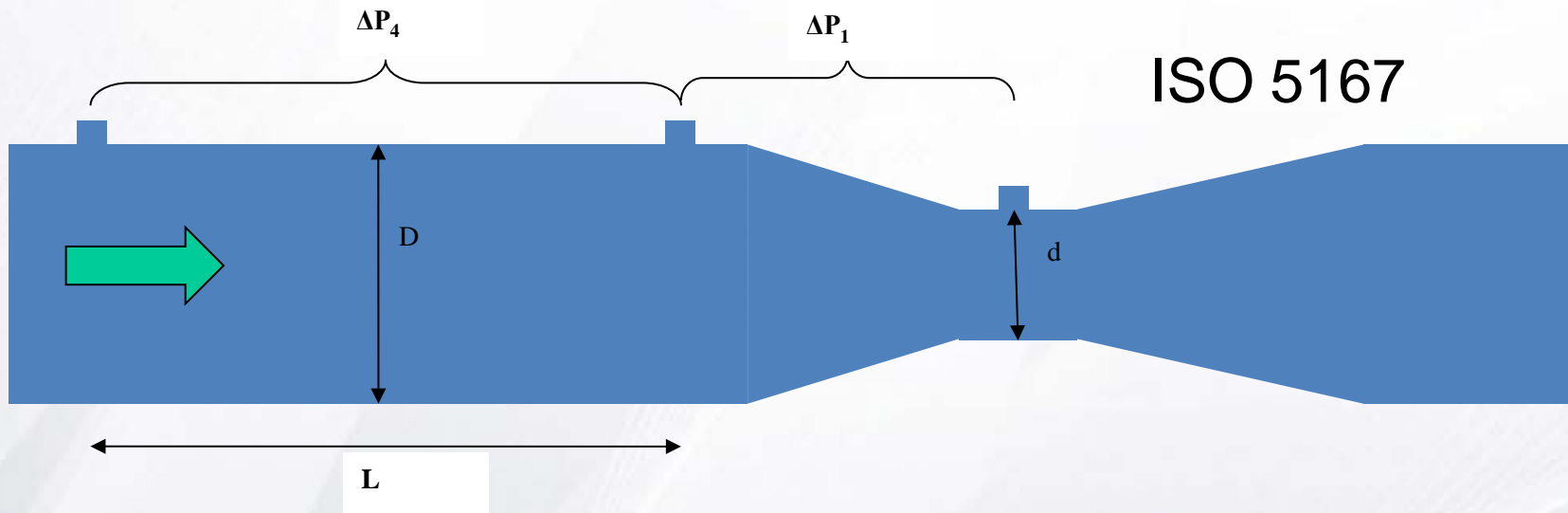
$$\frac{p_1}{\rho g} + \frac{u_1^2}{2g} + z_1 = \frac{p_2}{\rho g} + \frac{u_2^2}{2g} + z_2 \quad \text{Bernoulli- inviscid fluid}$$



$$Q_v = C_d \cdot \frac{\pi \cdot d^2}{4} \cdot \frac{1}{\sqrt{(1-\beta^4)}} \cdot \sqrt{\frac{2 \cdot (P_1 - P_2)}{\rho}} \quad \text{DP meter equation}$$

$$\lambda = \frac{2\Delta P_4 \cdot D}{\rho \cdot L \cdot u_D^2} \quad \text{Darcy-Weisbach}$$

Theory (2)

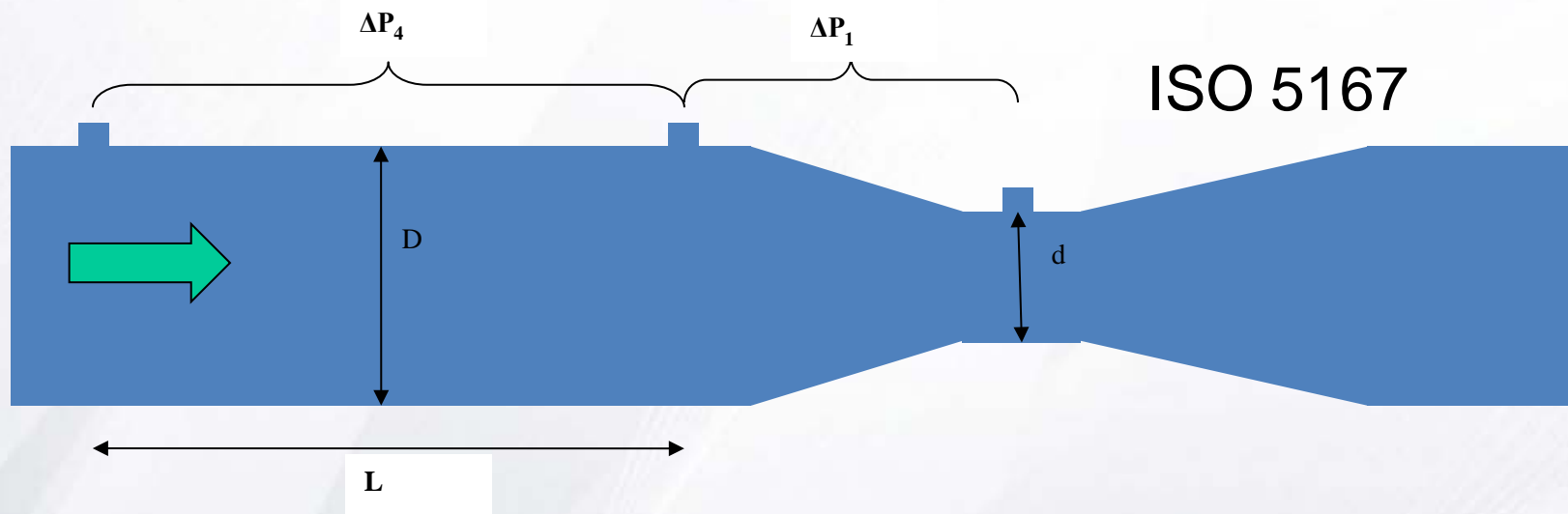


$$u_D^2 = \frac{2 \cdot C_d^2 \cdot \beta^4 \cdot \Delta P_1}{\rho(1 - \beta^4)}$$

Substitute into

$$\lambda = \frac{2\Delta P_4 \cdot D}{\rho \cdot L \cdot u_D^2}$$

Theory (3)



$$\lambda = \frac{\Delta P_4}{\Delta P_1} \cdot \frac{C_g}{C_d^2} \quad \text{where} \quad C_g = \frac{D(1 - \beta^4)}{L \cdot \beta^4}$$

**Independent
of ρ and μ**

Theory (4)

Convert λ to Reynolds number from theory

$$\text{Re} = \frac{64}{\lambda} \quad \text{Or} \quad \frac{1}{\sqrt{\lambda}} = -2 \log \left(\frac{\varepsilon}{3.7D} + \frac{2.51}{\text{Re} \sqrt{\lambda}} \right)$$

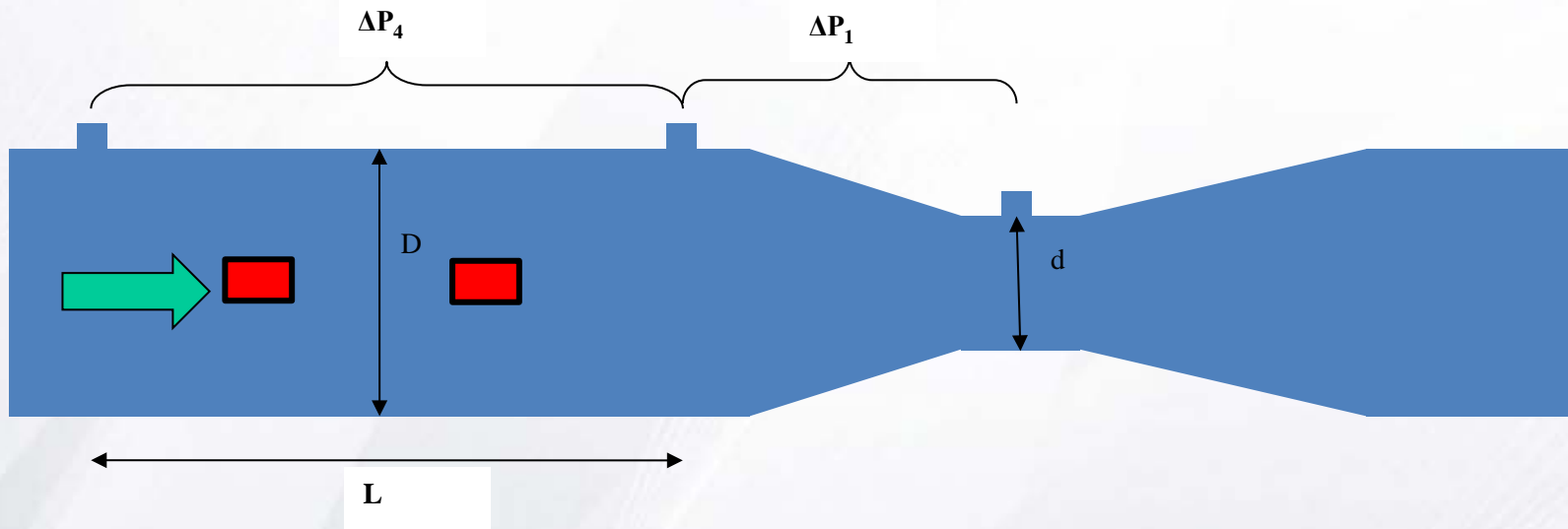
Since C_d as a function of Re is well known. We can iterate to find solution for C_d , λ and Re starting with a guessed C_d .

To calculate the flowrate, we still need ρ . Is there some way to calculate ρ in line? Yes, from the friction factor Eqn

$$\rho = \frac{2\Delta P_4 \cdot D}{\lambda \cdot L \cdot u_D^2}$$

But u_D is unknown...

Theory (5)

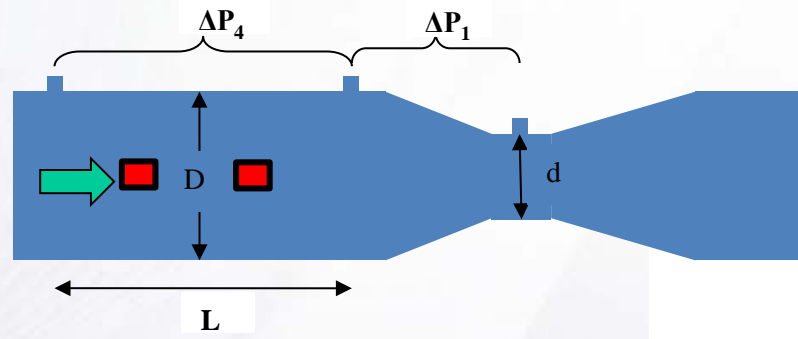


u_D can be obtained by adding secondary technology such as Clamp-on USM

$$\rho = \frac{2\Delta P_4 \cdot D}{\lambda \cdot L \cdot u_D^2}$$

$$\mu = \frac{\rho \cdot u_D \cdot D}{\text{Re}}$$

Theory Recap



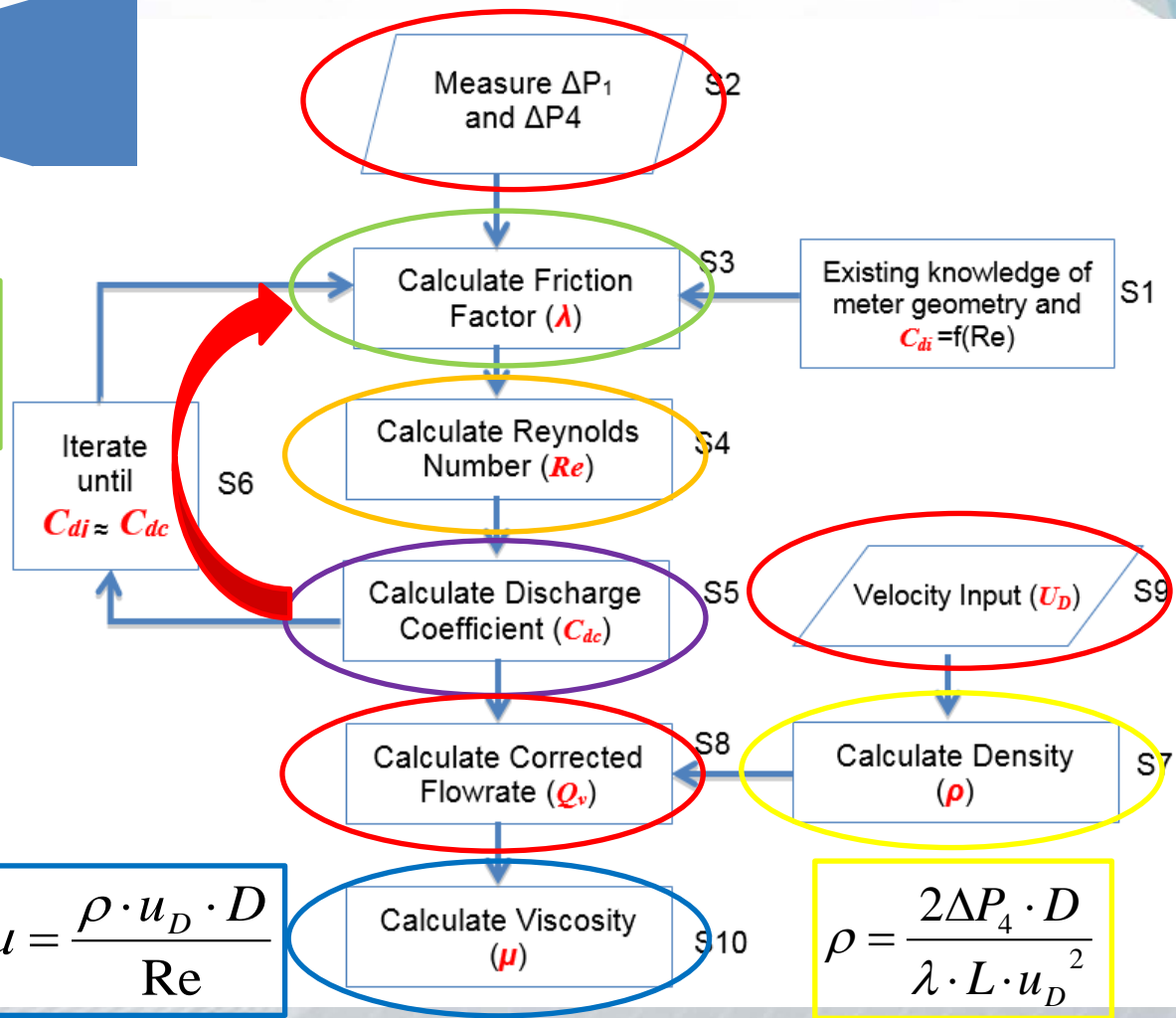
$$\lambda = \frac{\Delta P_4}{\Delta P_1} \cdot \frac{C_g}{C_d^2} \text{ where } C_g = \frac{D(1-\beta^4)}{L \cdot \beta^4}$$

$$Re = \frac{64}{\lambda} \text{ or from } \frac{1}{\sqrt{\lambda}} = -2 \log \left(\frac{\epsilon}{3.7D} + \frac{2.51}{Re \sqrt{\lambda}} \right)$$

$$C_d = f(Re)$$

$$\mu = \frac{\rho \cdot u_D \cdot D}{Re}$$

$$\rho = \frac{2\Delta P_4 \cdot D}{\lambda \cdot L \cdot u_D^2}$$



NEL Heavy Oil Facility

UK National Standard Oil facility:

Flow up to 200 l/s

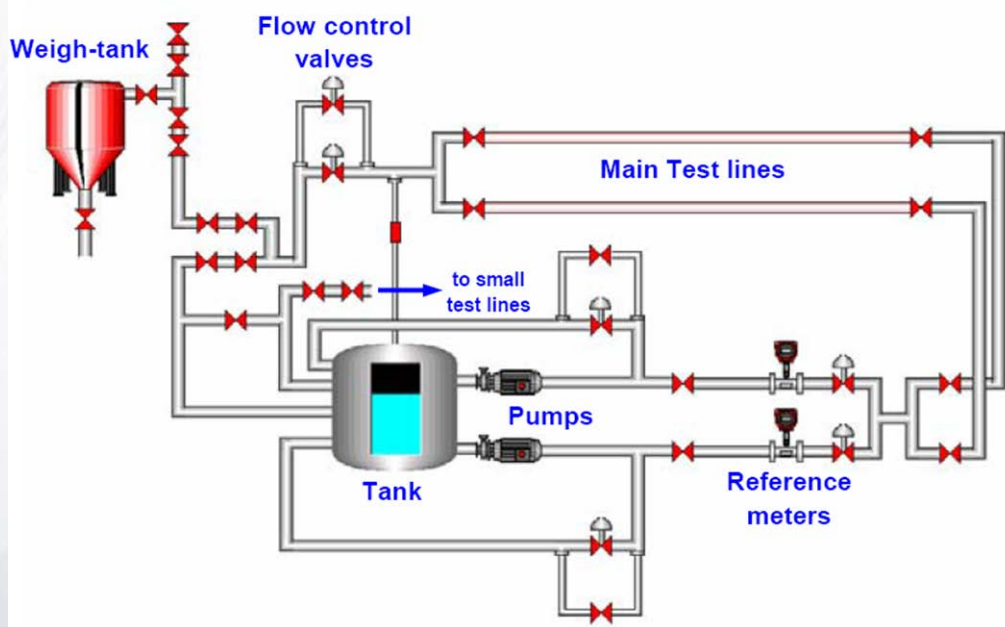
Oil viscosity from 1.5 to 1500 cSt

Test temperature up to 60°C

Temp control with $\pm 1^\circ\text{C}$

Test pressure up 10 bar

Pipe sizes 0.5" to 8"



Overall uncertainty in ref flow $\pm 0.03\%$ ([Gravimetric](#))

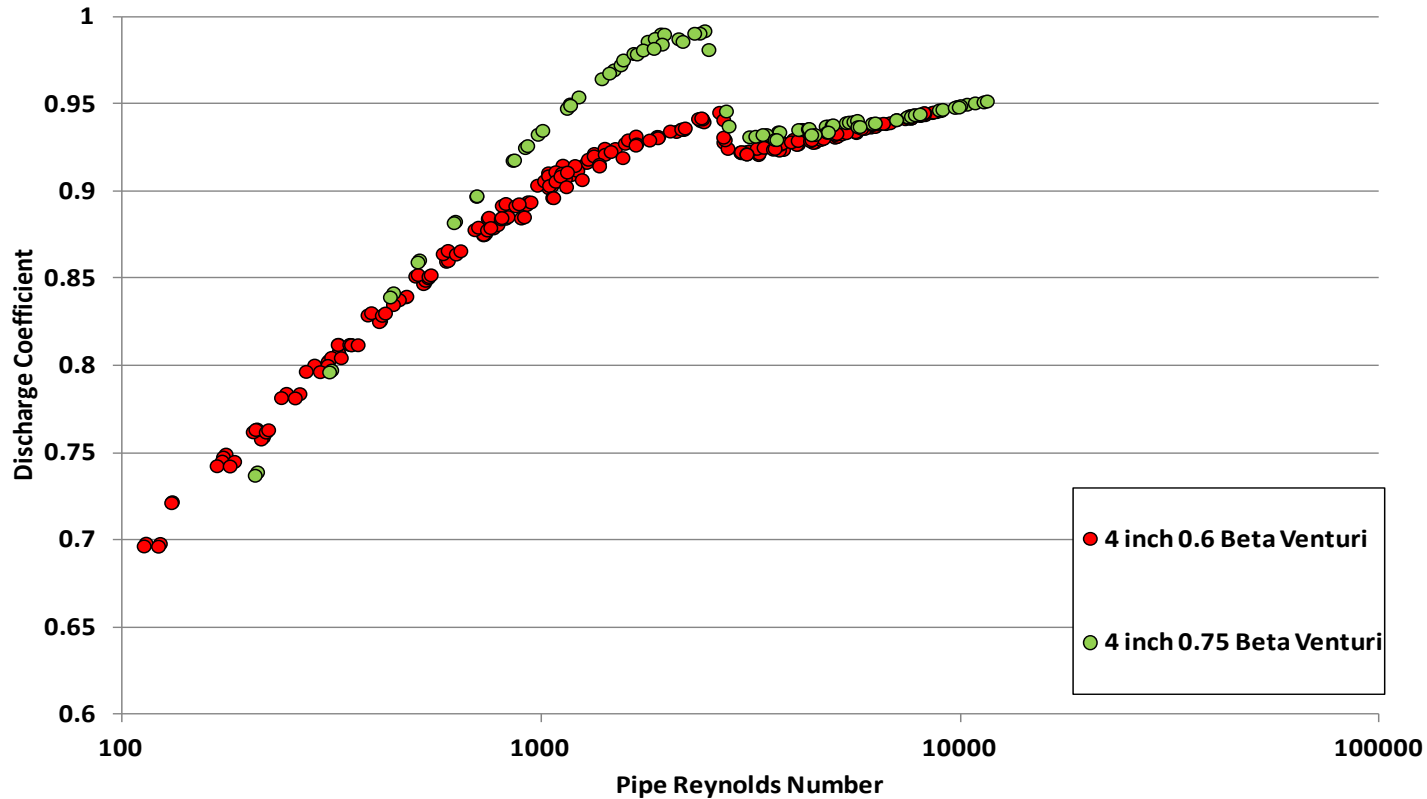
Overall uncertainty in ref flow $\pm 0.08\%$ ([Reference meters](#))

Test Meters

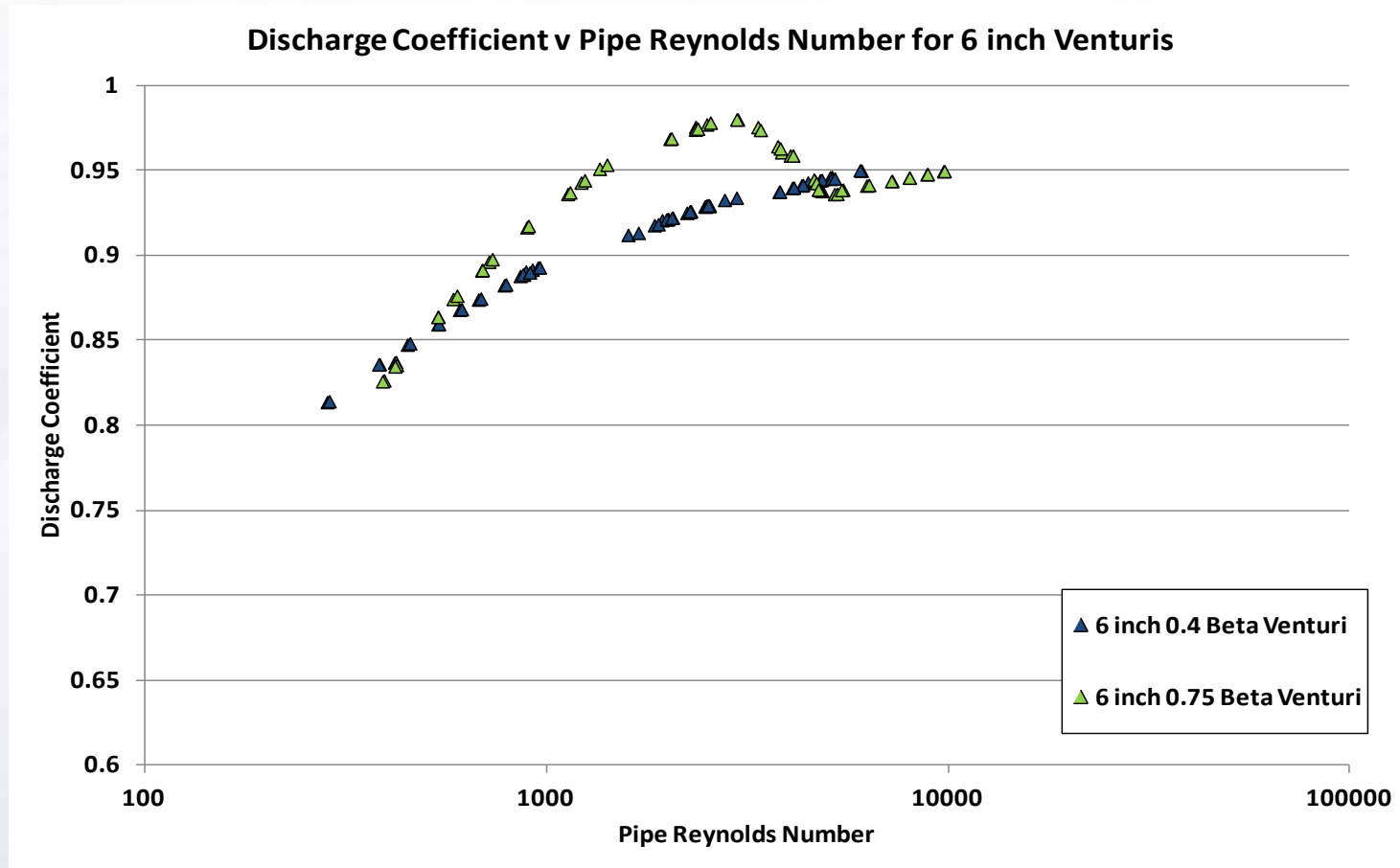
Meter Number	Meter Type	Nominal Size (in)	Nominal Beta	Upstream Length (m)	Min Re	Max Re
1	Venturi	8	0.4	2.027	94	8268
2	Venturi	8	0.6	2.027	84	9213
3	Venturi	6	0.4	1.540	285	6081
4	Venturi	6	0.75	1.540	391	9781
6	Venturi	4	0.6	1.022	88	11404
7	Venturi	4	0.75	1.022	208	14600
8	QE Orifice	8	0.45	2.393	647	9140
9	QE Orifice	8	0.6	2.393	240	7681

4 inch Venturi Results

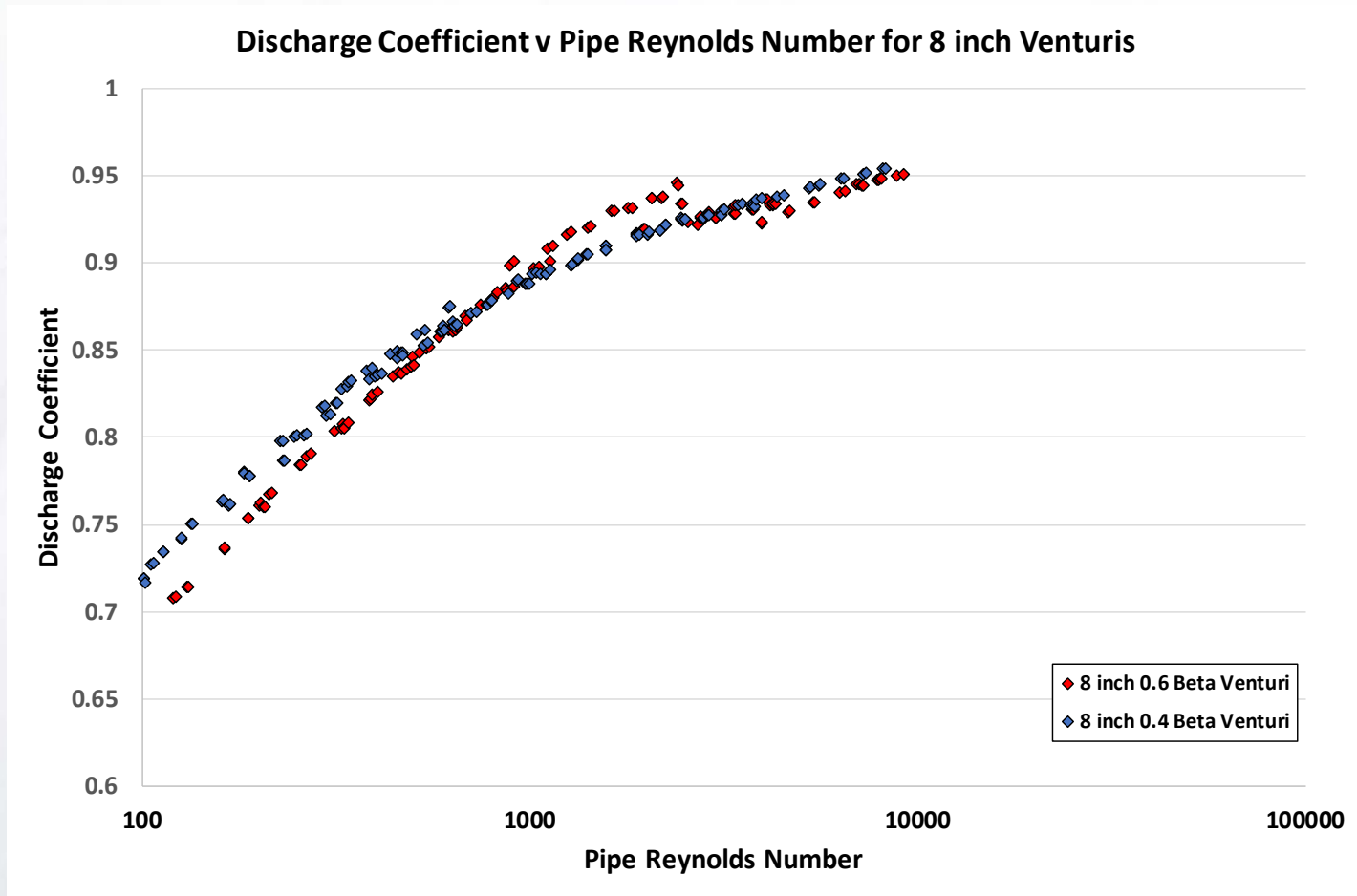
Discharge Coefficient v Pipe Reynolds Number for 4 inch Venturis



6 inch Venturi Results

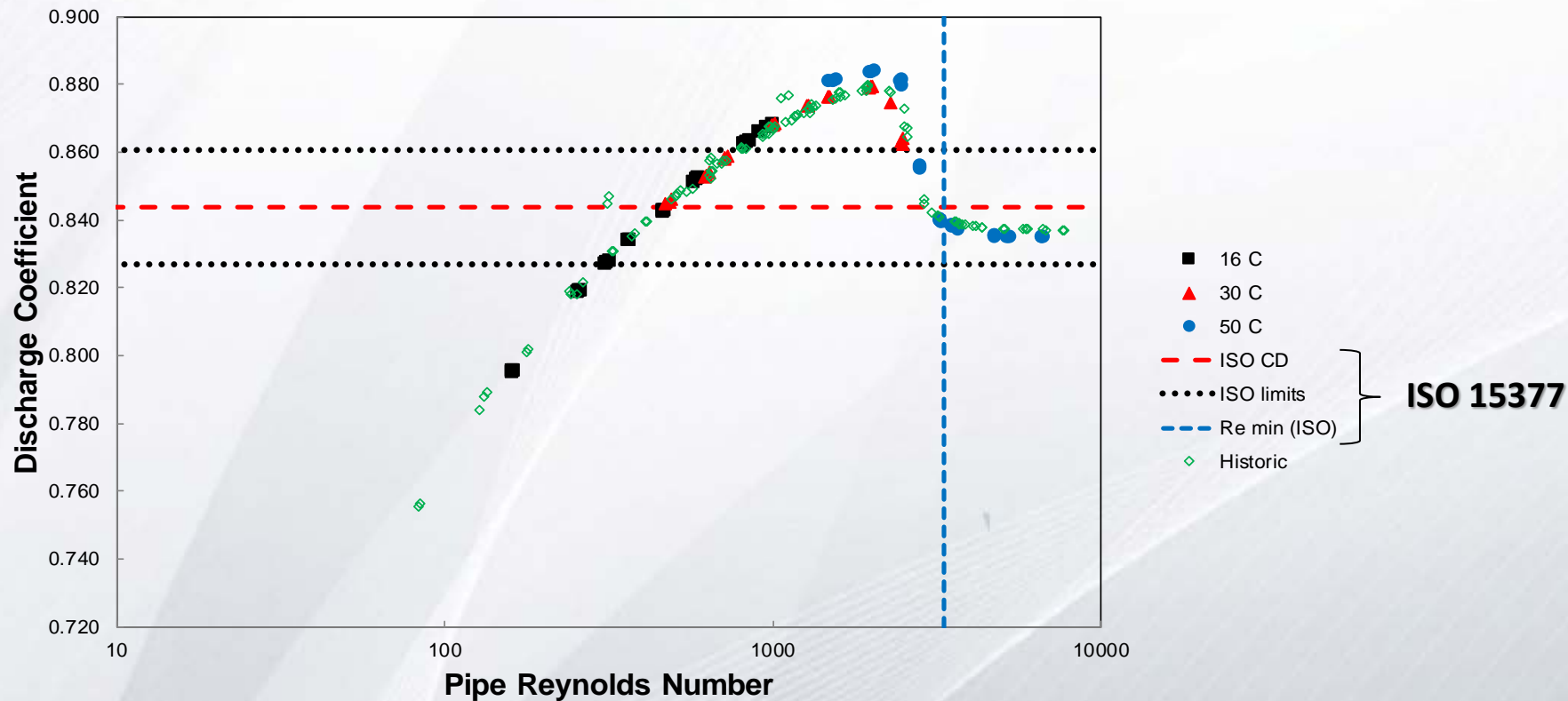


8 inch Venturi Results



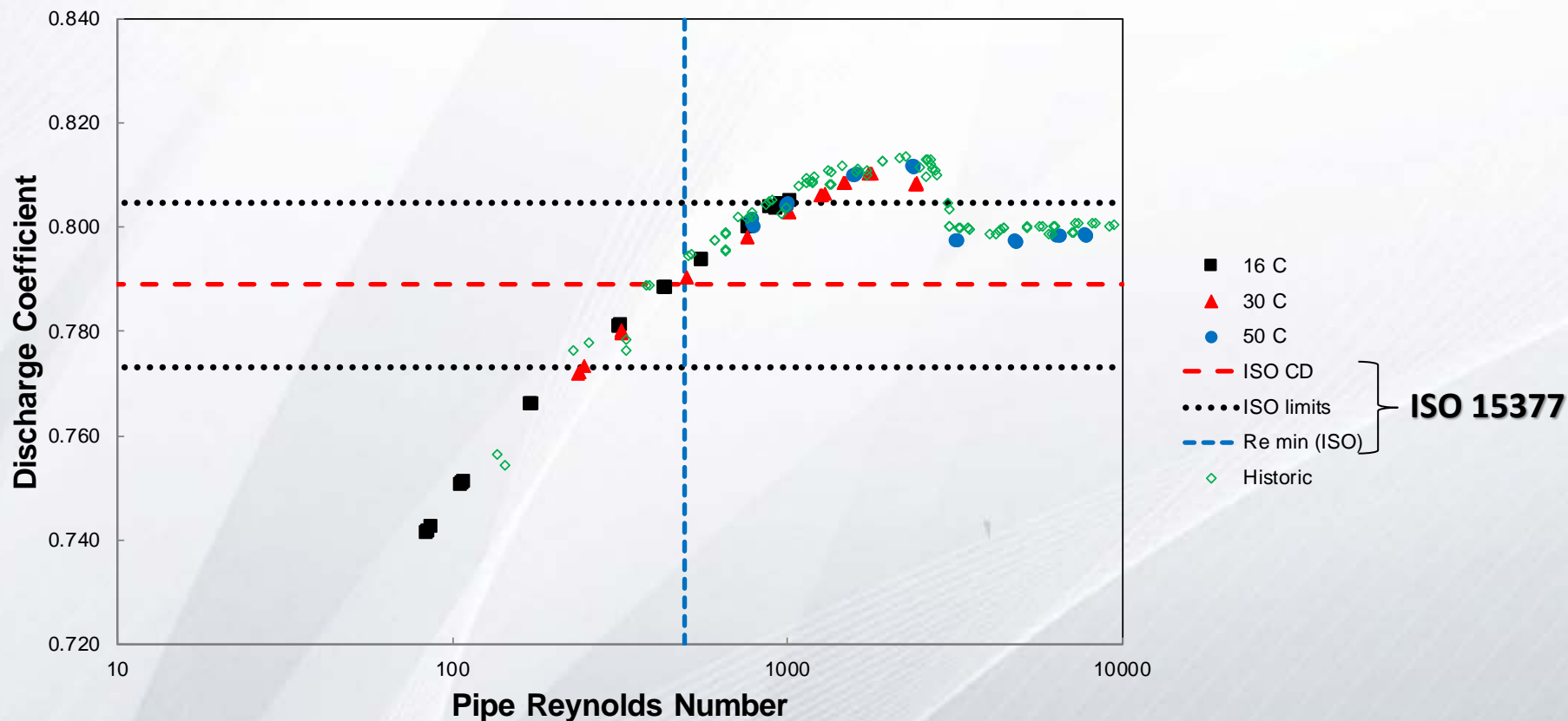
8 inch QE Orifice Results

Discharge Coefficient v Pipe Reynolds Number for 8 inch Quadrant Edge Orifice Plate $\beta=0.6$



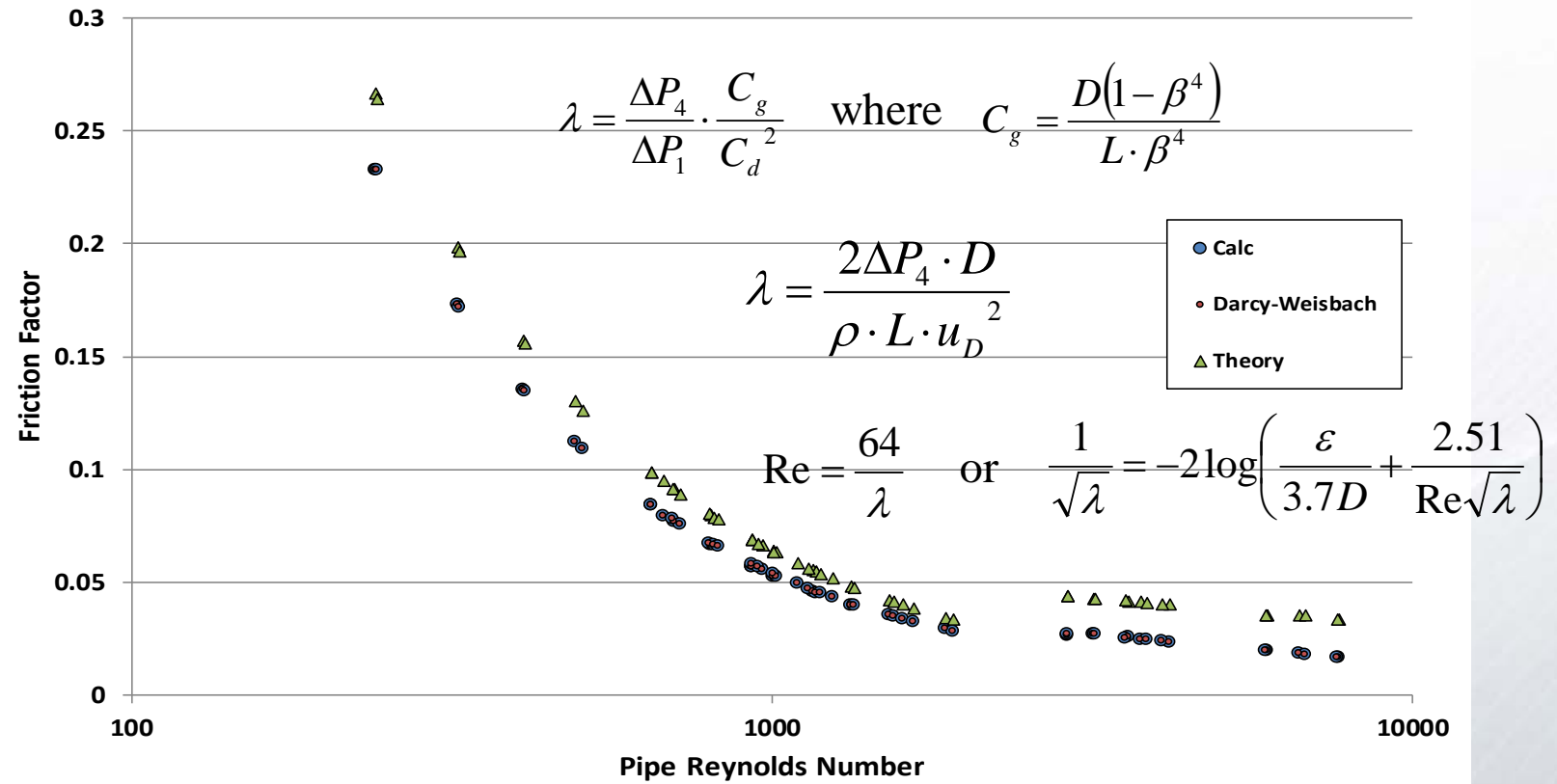
8 inch QE Orifice Results

Discharge Coefficient v Pipe Reynolds Number for 8 inch
Quadrant Edge Orifice Plate $\beta=0.45$



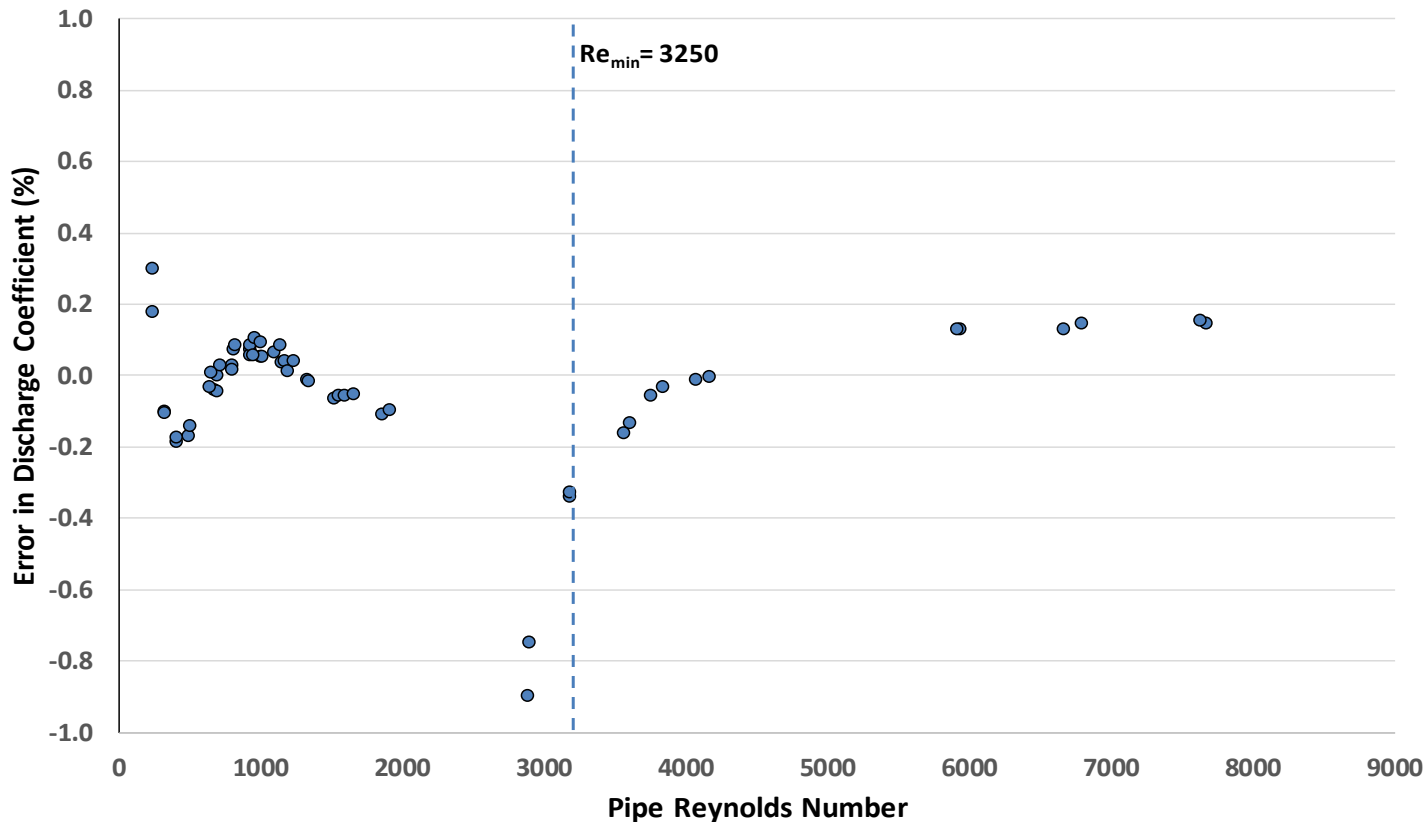
Applying New Method

Friction Factor v Pipe Reynolds Number



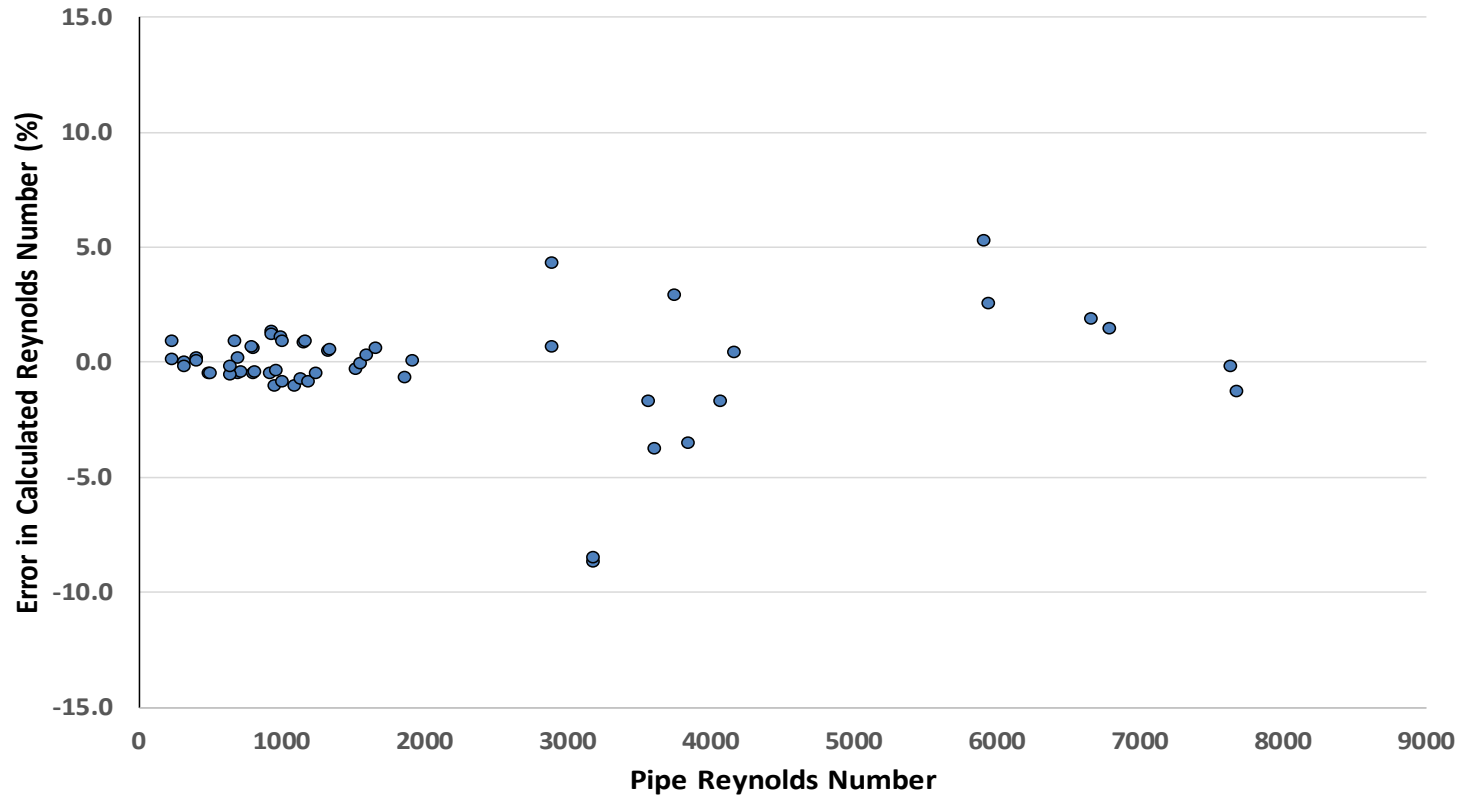
New Method (using input ρ)

Error in Discharge Coefficient v Pipe Reynolds Number



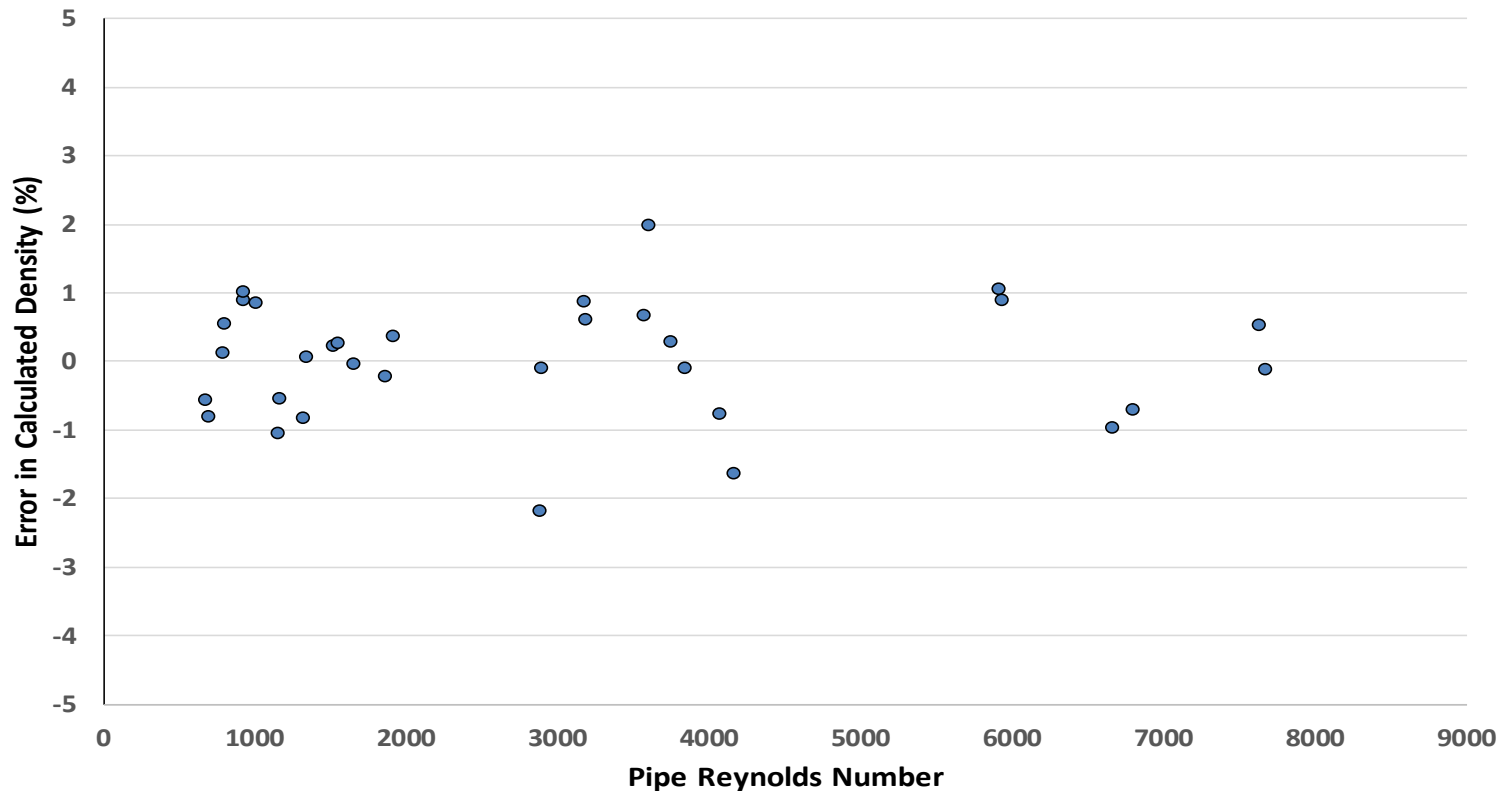
New Method (using input ρ)

Error in Calculated Reynolds Number v Pipe Reynolds Number



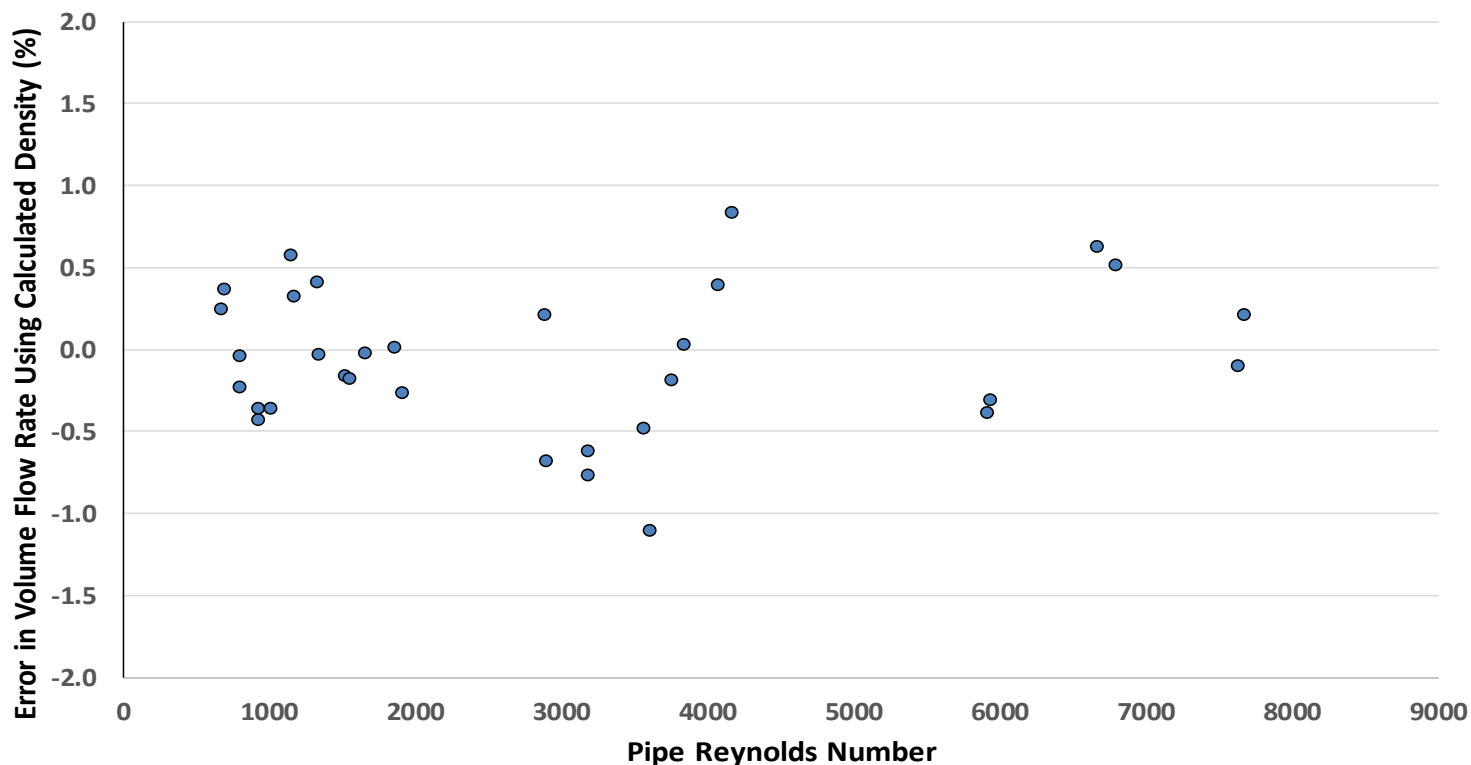
New Method (calculated ρ)

Error in Calculated Density v Pipe Reynolds Number



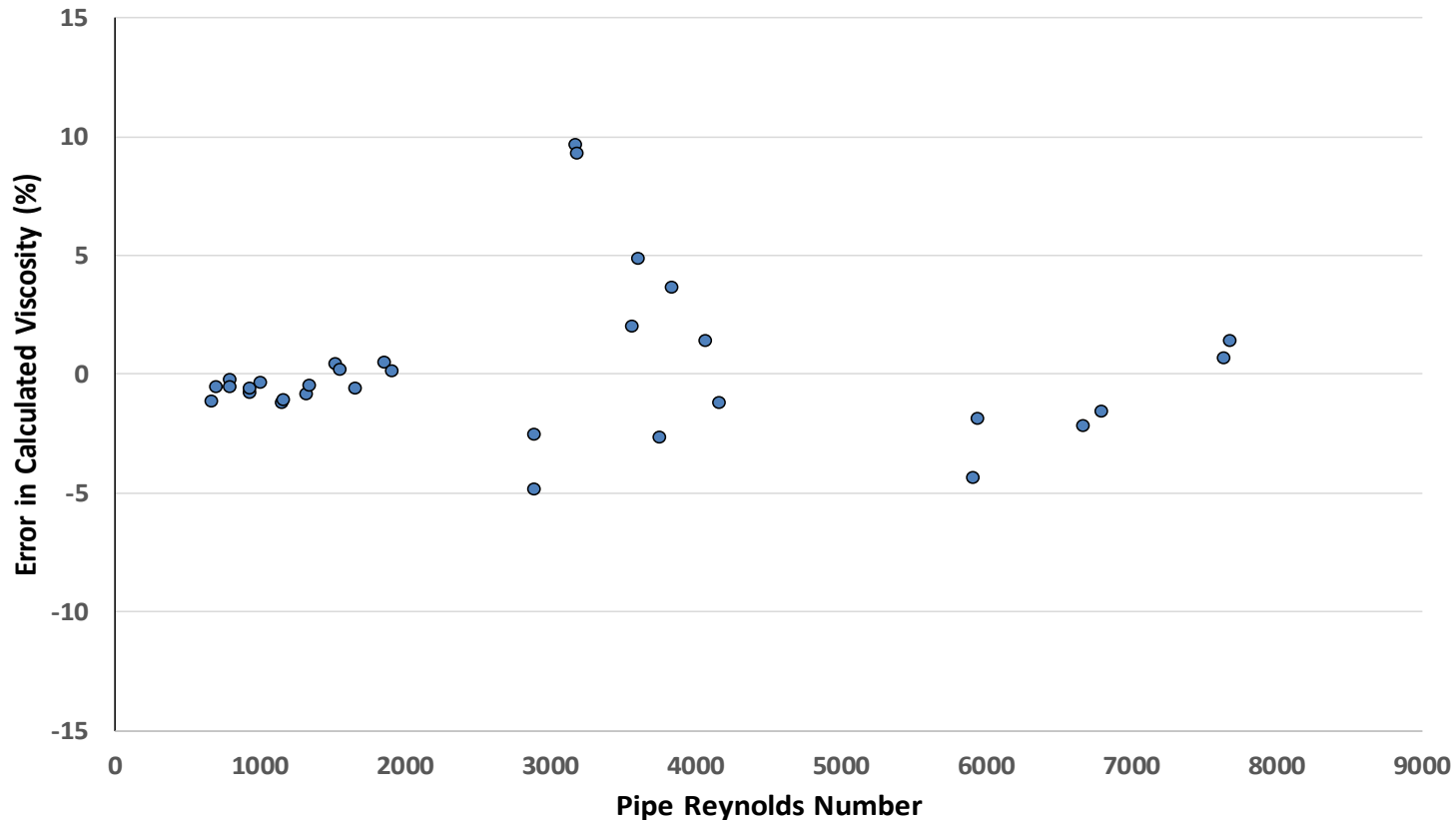
New Method (using calculated ρ)

Error in Volume Flow Rate Using Calculated Density v Pipe Reynolds Number



New Method (using calculated ρ)

Error in Calculated Viscosity v Pipe Reynolds Number



Conclusions

- A new method is presented to show the potential of DP meters for measurements in low Re applications.
- Initial limited testing have shown promising results:

Re: within $\pm 5\%$

Cd: within $\pm 0.2\%$

Vol flow: within $\pm 0.2\%$ (using known density)

- By using additional measurement technology we can get:

Density: within $\pm 2\%$

Vol flow: within $\pm 1\%$ (using calc density)

Viscosity: within $\pm 5\%$

However more work is required to develop this method further and to investigate the influence of several parameters

Future Work

- Investigate the influence of following parameters:
 - Influence of upstream length and installation effects
 - Influence of fluid properties and fluid condition
 - Influence of meter geometry
- Combine the method with other measurement technologies
- Developing new standard equation for low Re and update relevant standards
- Calculation of uncertainty

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Thanks for Attention